

CFD ANALYSIS OF SMALL HYDRO PLANT TURBINES: CASE STUDIES

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ABSTRACT

Performance analysis of Small Hydro Turbine is an essential practice to ensure that Small Hydro Plant performance is still at an acceptable level. In addition, performance testing is required as a part of programs to improve the efficiency, output, and economic performance of small hydro plants along with Optimization which has been an important focus in recent years. This study deals with three different Small Hydro Turbines of various manufacturer or projects. Comparison between simulation results and project/manufacturer data reveals a good agreement. To the best of the author's knowledge these novel approach for CFD analysis of Small Hydro Turbines together using DSS Solidworks Flow Simulation (FloEFD) is absent in renewable energy or fluid mechanics literature due to its assessment complexity.

Keywords: CFD, turbomachines, pelton wheel, Francis turbine, kaplan turbine.

1. INTRODUCTION

Hydro power projects are classified as large, medium and small (renewable energy) hydro projects based on their sizes. Ranging from 10MW to 50 MW various countries have various size criteria to classify small hydro projects. Hydro power plants of 25MW or below capacity are classified as small hydro in India. It is further classified into pico-hydro (5kW or below), micro hydro (6kW - 100kW), mini hydro (101kW-2MW) and small hydro (2-25MW) plants.

Various potential power generation sites are found in existing water infrastructures and delivery networks. Common uses are at water supply to transmission locations, tank fill locations, residential zones with elevation changes, and at large commercial areas. Most existing water infrastructure sites have pressure reducing valves already installed; they simply need the valves replaced with energy recovery turbines. [1-50]

2. METHODOLOGY ADOPTED

The purpose of this study was to check three types of small hydro plant turbines efficiency at various heads. The experiment and analyse were carried out to find the various parameters of the turbine, as well as the overall efficiency and compare them with manufacturer or project data (not detailed due to company privacy policy). [51-60]

3. THEORY AND CALCULATION

The governing equations for fluid flow and heat transfer are the Navier-Stokes or momentum equations and the First Law of Thermodynamics or energy equation. The analyses of these 03 cases (Pelton Wheel, Francis Turbine, Kaplan Turbine) reveal that significant improvement could be obtained by applying the proper arrangements of vanes, draft tubes etc.

The Favre-averaged Navier-Stokes equations are used, where time-averaged effects of the flow turbulence on the flow parameters are considered, whereas the largescale, time-dependent phenomena are taken into account directly. Through this procedure, extra terms known as the Reynolds stresses appear in the equations for which additional information must be provided.

To close this system of equations, DSS SOLIDWORKS Flow Simulation (FloEFD) employs transport equations for the turbulent kinetic energy and its dissipation rate, using most popular the k- ϵ model [61-77].

The purpose of these performance analysis of Pelton Wheel, Francis Turbine and Kaplan Turbine is to determine: power generated, velocity, pressure, various turbulence parameter distributions in runner for future work (on fatigue analysis). The governing PDEs can be written as:

Table-1. C	CFD analysis	- governing	equations.
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Continuity Equation: $\frac{\partial p}{\partial t} + \frac{\partial p u}{\partial x} + \frac{\partial p v}{\partial y} + \frac{\partial p w}{\partial z} = 0$ X-Momentum Equation: $p\frac{\partial u}{\partial t} + p u \frac{\partial u}{\partial x} + p v \frac{\partial u}{\partial y} + p w \frac{\partial u}{\partial z}$ $= p g_x - \frac{\partial p}{\partial x} + \frac{\partial}{\partial x} [2 \mu \frac{\partial u}{\partial x}] + \frac{\partial}{\partial y} [\mu (\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x})] + \frac{\partial}{\partial z} [\mu (\frac{\partial u}{\partial z} + \frac{\partial w}{\partial x})]$ $+ S_{00} + S_{DR}$ Y-Momentum Equation: $p\frac{\partial v}{\partial t} + p u \frac{\partial v}{\partial x} + p v \frac{\partial v}{\partial y} + p w \frac{\partial v}{\partial z}$ $= p g_y - \frac{\partial p}{\partial y} + \frac{\partial}{\partial x} [\mu (\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x})] + \frac{\partial}{\partial y} [2 \mu \frac{\partial v}{\partial y}] + \frac{\partial}{\partial z} [\mu (\frac{\partial v}{\partial z} + \frac{\partial w}{\partial y})]$ $+ S_{00} + S_{DR}$ Z-Momentum Equation: $p\frac{\partial w}{\partial t} + p u \frac{\partial w}{\partial x} + p v \frac{\partial w}{\partial y} - p w \frac{\partial w}{\partial z}$ $= p g_z - \frac{\partial p}{\partial z} + \frac{\partial}{\partial x} [\mu (\frac{\partial w}{\partial z} + \frac{\partial w}{\partial y})] + \frac{\partial}{\partial y} [\mu (\frac{\partial v}{\partial z} + \frac{\partial w}{\partial y})] + \frac{\partial}{\partial z} [2 \mu \frac{\partial w}{\partial z}]$ $+ S_{00} + S_{DR}$

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These studies present Computational Fluid Dynamics (CFD) or performance analysis of Pelton Wheel, Francis Turbine and Kaplan Turbine for Small Hydro Projects.

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4A. CASE STUDY-1: PELTON WHEEL

This study presents Computational Fluid Dynamics (CFD) analysis of Pelton Wheel using DSS Solidworks Flow Simulation (FloEFD). The purpose of performance analysis is to determine torque generated by the turbine, velocity, pressure, various turbulence parameter distributions in bucket.

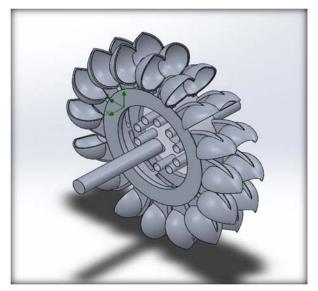


Figure-1. Pelton wheel - 3D model of runner.

The CFD analysis is carried out on model size Pelton runner reduced scale to minimize computational time, effort and cost. The operating conditions for model size runner are selected in accordance with IEC 60193 and IEC 1116 (not detailed here due to company privacy policy) as shown below.

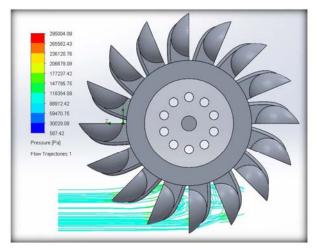


Figure-2. Pelton wheel 3D model - pressure plot.

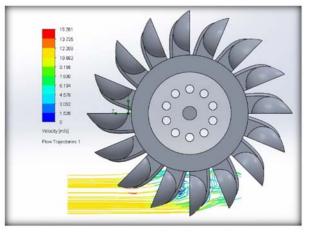


Figure-3. Pelton wheel 3D model - velocity plot.

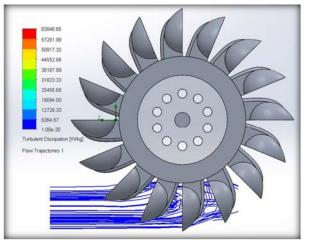


Figure-4. Pelton wheel 3D model - turbulence dissipation plot.

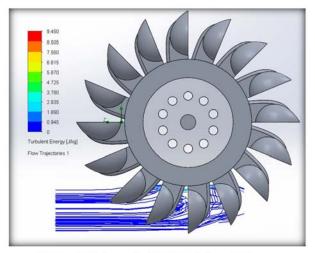


Figure-5. Pelton wheel 3D model - turbulence energy plot.

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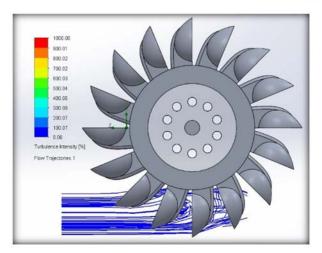


Figure-6. Pelton wheel 3D model - turbulence intensity plot.

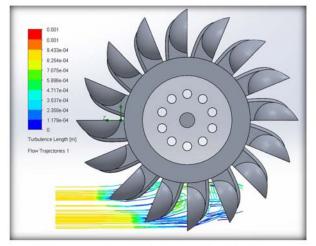


Figure-7. Pelton wheel 3D model - turbulence length plot.

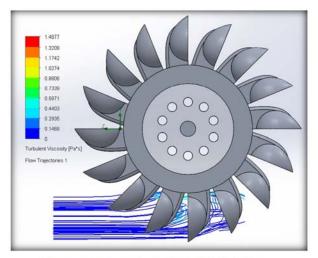


Figure-8. Pelton wheel 3D model - turbulence viscosity plot.

Table-2. Pelton wheel simulation data - min/max values of
various parameters.

Parameter	Minimum	Maximum
Absolute Humidity [kg/m^3]	377.13	998.74
Density (Fluid) [kg/m ³]	377.17	998.84
Mass Fraction of Dissolved gas []	1.0000e-04	1.0000e-04
Mass Fraction of Vapour []	0	1.0000000
Mass Fraction of Water []	0.9999	0.9999
Pressure [Pa]	587.42	295004.09
Temperature [K]	289.81	293.64
Temperature (Fluid) [K]	289.81	293.64
Velocity [m/s]	0	15.261
Velocity (X) [m/s]	-10.784	12.297
Velocity (Y) [m/s]	-12.559	8.077
Velocity (Z) [m/s]	-15.147	6.348
Volume Fraction of Vapour []	0	0.9999378
Mach Number []	0	5.03
Velocity RRF [m/s]	0	15.261
Velocity RRF (X) [m/s]	-10.784	12.297
Velocity RRF (Y) [m/s]	-12.559	8.077
Velocity RRF (Z) [m/s]	-15.147	6.348
Vorticity [1/s]	0.28	9203.50
Relative Pressure [Pa]	-100737.58	193679.09
Shear Stress [Pa]	0	513.48
Bottleneck Number []	7.8217525e-16	1.0000000
Heat Transfer Coefficient [W/m*2/K]	0	0
ShortCut Number []	1.1377939e-15	1.0000000
Surface Heat Flux [W/m ²]	0	0
Surface Heat Flux (Convective) [W/m ²]	0	0
Turbulence Intensity [%]	0.08	1000.00
Turbulence Length [m]	0	0.001
Turbulent Dissipation [W/kg]	1.00e-20	63646.65
Turbulent Energy [J/kg]	0	9.450
Turbulent Time [s]	0	0.351
Turbulent Viscosity [Pa*s]	0	1.4677
Acoustic Power [W/m^3]	0	2748.031
Acoustic Power Level [dB]	0	154.39

Various Min-Max Parameter values (table) for Pelton Wheel is shown above.

4B. CASE STUDY-2: FRANCIS TURBINE

This study presents Computational Fluid Dynamics (CFD) analysis of Francis Turbine using DSS Solidworks Flow Simulation (FloEFD). The purpose of performance analysis is to determine power generated by the turbine, velocity, pressure, various turbulence parameter distributions in vanes/blades for further work (as fatigue analysis).

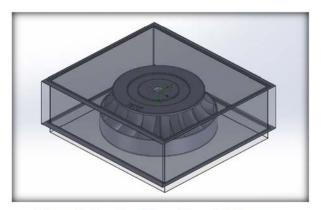


Figure-9. Francis turbine - 3D model of runner.

The CFD analysis is carried out on model size Francis Turbine runner reduced scale to minimize computational time, effort and cost. The operating conditions for model size runner are selected in accordance with IEC 60193 and IEC 1116 (not detailed here due to company privacy policy) as shown below.

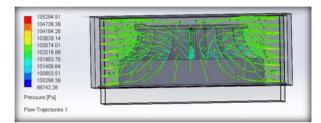
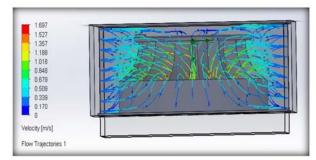
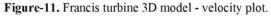


Figure-10. Francis turbine 3D model - pressure plot.





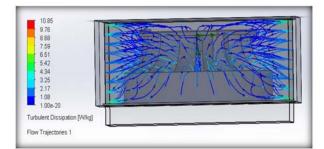


Figure-12. Francis turbine 3D model - turbulence dissipation plot.

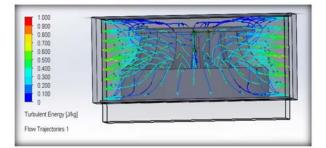


Figure-13. Francis turbine 3D model - turbulence energy plot.

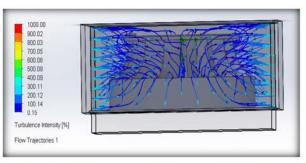


Figure-14. Francis turbine 3D model - turbulence intensity plot.

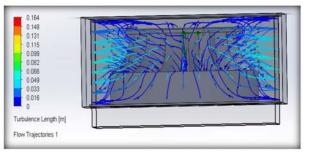


Figure-15. Francis turbine 3D model - turbulence length plot.

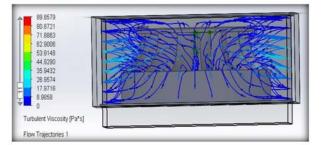


Figure-16. Francis turbine 3D model - turbulence viscosity plot.



 Table-3. Francis turbine simulation data - min/max values of various parameters.

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Parameter	Minimum	Maximum
Absolute Humidity [kg/m^3]	998.32	998.32
Density (Fluid) [kg/m ³]	998.42	998.42
Mass Fraction of Dissolved gas []	1.0000e-04	1.0000e-04
Mass Fraction of Vapour []	0	0
Mass Fraction of Water []	0.9999	0.9999
Pressure [Pa]	99743.26	105294.51
Temperature [K]	293.19	293.20
Temperature (Fluid) [K]	293.19	293.20
Velocity [m/s]	0	1.697
Velocity (X) [m/s]	-1.416	1.414
Velocity (Y) [m/s]	-0.909	1.335
Velocity (Z) [m/s]	-1.350	1.272
Volume Fraction of Vapour []	0	0
Mach Number []	0	1.63e-03
Velocity RRF [m/s]	0	1.697
Velocity RRF (X) [m/s]	-1.416	1.414
Velocity RRF (Y) [m/s]	-0.909	1.335
Velocity RRF (Z) [m/s]	-1.350	1.272
Vorticity [1/s]	8.35e-03	63.24
Relative Pressure [Pa]	-1581.74	3969.51
Shear Stress [Pa]	0	9.10
Bottleneck Number []	5.4192219e-07	1.0000000
Heat Transfer Coefficient [W/m ² /K]	0	0
ShortCut Number []	0.0000721	1.0000000
Surface Heat Flux [W/m ²]	0	0
Surface Heat Flux (Convective) [W/m^2]	-5.160e+08	3.244e+09
Turbulence Intensity [%]	0.15	1000.00
Turbulence Length [m]	0	0.164
Turbulent Dissipation [W/kg]	1.00e-20	10.85
Turbulent Energy [J/kg]	0	1.000
Turbulent Time [s]	0	1.704
Turbulent Viscosity [Pa*s]	0	89.8579
Acoustic Power [W/m^3]	0	6.714e-13
Acoustic Power Level [dB]	0	0

Various Min-Max Parameter values (table) for Francis Turbine is shown above.

4C. CASE STUDY-3: KAPLAN TURBINE

This study presents Computational Fluid Dynamics (CFD) analysis of Kaplan Turbine using DSS Solidworks Flow Simulation (FloEFD). The purpose of performance analysis is to determine torque generated by the turbine, velocity, pressure, various turbulence parameter distributions in vanes/blades for further work.

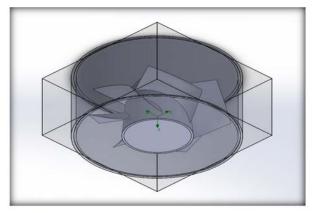


Figure-17. Kaplan turbine - 3D model of runner.

The CFD analysis is carried out on model size Kaplan Turbine runner reduced scale to minimize computational time, effort and cost. The operating conditions for model size runner are selected in accordance with IEC 60193 and IEC 1116 (not detailed here due to company privacy policy) as shown below.



Figure-18. Kaplan turbine 3D model - pressure plot.

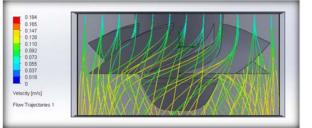


Figure-19. Kaplan turbine 3D model - velocity plot.

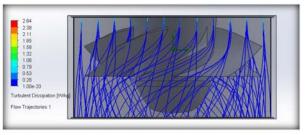


Figure-20. Kaplan turbine 3D model - turbulence dissipation plot.

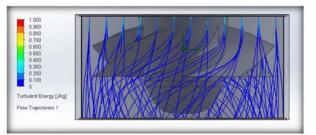


Figure-21. Kaplan turbine 3D model - turbulence energy plot.



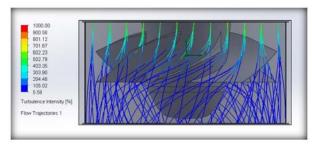


Figure-22. Kaplan turbine 3D model - turbulence intensity plot.

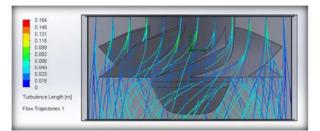


Figure-23. Kaplan turbine 3D model - turbulence length plot.

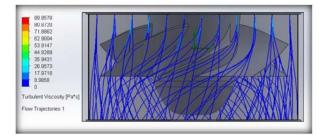


Figure-24. Kaplan turbine 3D model - turbulence viscosity plot.

Table-4. Kaplan turbine simulation data - min/max values
of various parameters.

Parameter	Minimum	Maximum
Absolute Humidity [kg/m ⁴ 3]	998.32	998.32
Density (Fluid) [kg/m ³]	998.42	998.42
Mass Fraction of Dissolved gas []	1.0000e-04	1.0000e-04
Mass Fraction of Vapour []	0	0
Mass Fraction of Water []	0.9999	0.9999
Pressure [Pa]	101307.86	101363.17
Temperature [K]	293.20	293.20
Temperature (Fluid) [K]	293.20	293.20
Velocity [m/s]	0	0.184
Velocity (X) [m/s]	-0.152	0.153
Velocity (Y) [m/s]	-0.147	0.031
Velocity (Z) [m/s]	-0.167	0.172
Volume Fraction of Vapour []	0	0
Mach Number []	0	1.77e-04
Velocity RRF [m/s]	0	0.184
Velocity RRF (X) [m/s]	-0.152	0.153
Velocity RRF (Y) [m/s]	-0.147	0.031
Velocity RRF (Z) [m/s]	-0.167	0.172
Vorticity [1/s]	9.28e-03	1.42
Relative Pressure [Pa]	-17.14	38.17
Shear Stress [Pa]	0	0.13
Bottleneck Number []	5.4580202e-08	1.0000000
Heat Transfer Coefficient [W/m ² /K]	0	0
ShortCut Number []	0.0005882	1.0000000
Surface Heat Flux [W/m ²]	0	0
Surface Heat Flux (Convective) [W/m ²]	-1.696e+08	3.570e+08
Turbulence Intensity [%]	5.58	1000.00
Turbulence Length [m]	0	0.164
Turbulent Dissipation [W/kg]	1.00e-20	2.64
Turbulent Energy [J/kg]	0	1.000
Turbulent Time [s]	0	16.470
Turbulent Viscosity [Pa*s]	0	89.8578
Acoustic Power [W/m^3]	0	4.684e-13
Acoustic Power Level [dB]	0	0

Various Min-Max Parameter values (table) for Kaplan Turbine is shown above.

5. RESULT AND DISCUSSIONS

Comparison between simulation results and experimental / manufacturer data for the Pelton Wheel reveals good agreementas shown below.



Figure-25A. Pelton wheel - experimental setup.

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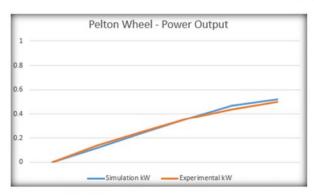


Figure-25B. Pelton wheel - output result comparison.

Comparison between simulation results and experimental / manufacturer data for the Francis Turbine reveals good agreement as shown below.



Figure-26A. Francis turbine - experimental setup.

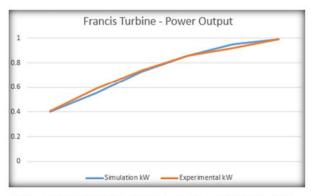


Figure-26B. Francis turbine - output result comparison.

Comparison between simulation results and experimental / manufacturer data for the Kaplan Turbine reveals good agreement as shown below.

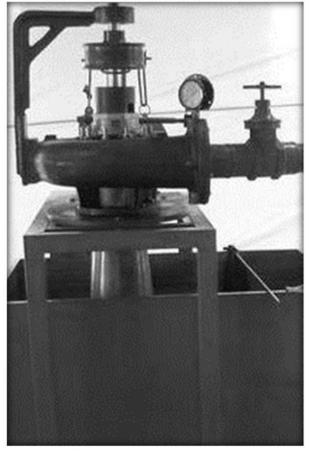


Figure-27A. Kaplan turbine - experimental setup.

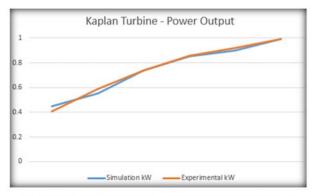


Figure-27B. Kaplan turbine - output result comparison.

6. CONCLUSIONS

Advantages of CFD models include their basis in the fundamental physics of fluid flow, a flexibility that allows the analysis of a huge range of boundary conditions, and the ability to design much more detailed geometries compared to algebraic or zone models. Disadvantages of CFD models include increased complexity, proper training of the tool user, greater computational capacity requirements of system, and a longer timeline between initiating a project and completing the necessary performance analysis.

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